DESCRIPTIONS

A RECIPROCATING SAW

Field of the invention

The present utility model relates to a reciprocating saw.

Background

In the prior art, a reciprocating saw comprises: a housing; a plunger in the housing for mounting a saw blade; a driving gear connecting with driving mechanism; a first eccentric shaft and a second eccentric shaft which all disposed on said driving gear; a link shaft which rear end portion connecting with said first eccentric shaft; a pushing member which rear end portion rotatablely connecting with said second eccentric shaft, a lifting member; a front end portion of said link shaft rotatablely connecting with a rear end portion of said plunger, said lifting member pivotally connecting with said housing; a first eccentric shaft drives a link shaft to move reciprocally so that the plunger is able to drive a saw blade move reciprocally fore and aft when the driving gear rotate. For saving energy, the saw blade is desired in a depress downwardly state from the workpiece when the saw blade move in the anti teeth direction, and the saw blade is desired in a slightly lifting upwardly state from the workpiece, for obtaining such a desirable solution, a blade orbital lifting device is supplied, a actuating member for blade lifting device in the conventional design actuating the blade lifting device by the pull way, in this kind structure, the moving direction of the actuating member is consistent with the moving direction of the plunger of the reciprocating saw so that caused a acute vibration of the reciprocating saw during using.

Invention Content

The object of present utility model is to provide a low vibration reciprocating saw.

The technical proposal of the present utility model is: a reciprocating saw, comprising: a housing; a plunger in the housing for mounting a saw blade; a driving gear connecting with driving mechanism; a first eccentric shaft] and a second eccentric shaft which all disposed on said driving gear; a link shaft which rear end portion connecting with said first eccentric shaft; a pushing member which rear end portion rotatablely connecting with said second eccentric shaft, a lifting member; a front end portion of said link shaft rotatablely connecting with a rear end portion of said plunger, said lifting member pivotally connecting with said housing; said lifting member contacting with a front end portion of said pushing member and also contacting with a sleeve bearing which disposed on a rear end portion of said plunger, a angle formed between lines through a axle center of a first eccentric shaft, a axle center of said driving gear and a axle center of a second eccentric shaft is an obtuse angle, said lines lies in a plane which perpendicular to said first eccentric shaft.

The advantages of the present utility model as following:

The angle between lines through a center of a first eccentric shaft, a rotating center of said driving gear and a rotating center of a second eccentric shaft is an obtuse angle so that said pushing member will move backward when said link shaft move forward, the movement of said pushing member in the direction opposite to the movement of said link shaft will eliminate the vibration which resulting from the reciprocating movements of said plunger.

BRIEF DESCRIPTION OF THE DRAWINGS

The accompanying drawings which are incorporated in, and constitute a part of, this specification illustrate one embodiment of the utility model and together with the description serve

to explain the principles of the utility model.

FIG. 1 shows a mainly sectional view of this utility model;

FIG. 2 shows a sectional view in line A-A of FIG. 1;

FIG. 3 shows a sectional view in line B-B of FIG. 1;

FIG. 4 shows a perspective view of said lifting member;

FIG. 5 shows a perspective view of said sleeve bearing;

In all the FIGS, the numbers respectively indicate to:

[1], a saw blade; [2], a spring; [3], a pivot; [4], a link shaft; [5], a first eccentric shaft; [6], a counterweight; [7], a driving gear; [8], a second eccentric shaft; [9], a pushing member; [10], a pivot for said link shaft; [11], a lifting member; [12], a sleeve bearing; [13], a housing; [14], a ball bearing; [15], a plunger; [16], a connecting pin; [17], transmitting gear shaft;

DESCRIPTION OF THE PREFERED EMBODIMENTS

See the accompanying Figs 1 and 2, A reciprocating saw, comprising: a housing[13]; a plunger [15] in the housing for mounting a saw blade [1]; a driving gear [7] connecting with driving mechanism; a first eccentric shaft [5] and a second eccentric shaft [8] which all disposed on said driving gear [7]; a link shaft [4] which rear end portion connecting with said first eccentric shaft [5]; a pushing member [9] which rear end portion rotatablely connecting with said second eccentric shaft [8], a lifting member [11]; a front end portion of said link shaft [4] rotatablely connecting with a rear end portion of said plunger [15], said lifting member [11] pivotally connecting with said housing [13] via a pivot [3]; said lifting member [11] contacting with a front end portion of said pushing member [9] and also contacting with a sleeve bearing [12] which disposed on the rear end portion of said plunger [15]. A front end portion of said pushing member push said lifting member [11] rotating about a pivot [3] in the clockwise direction when said pushing member move forward actuating by the rotation of said driving gear, so that said lifting member [11] lift said sleeve bearing [12] upwardly which bring a rear end portion of said plunger [15] move upwardly together, the upward movement of said plunger [15] cause said plunger [15] swing about a ball bearing [14] so that said saw blade [1] mounting on the front end portion of said plunger [15] synchronically move downwardly, this downwardly movement is propitious to the cutting operation During the saw blade return, upon the depressing of the spring [2], said lifting member [11] rotates about the pivot [3] in the anticlockwise direction, the movements of the rear end portion of said plunger [15] is in the downward direction, so the plunger [15] swing about the ball bearing [14] in the clockwise direction and the saw blade [1] lifted upwardly slightly away from the workpiece, then reducing the friction between the saw blade [1] and the workpiece so energy be saved. As illustrated in Fig 3, a angle formed between lines through the axle center of a first eccentric shaft [5], the axle center of said driving gear [7] and the axle center of a second eccentric shaft [8] is an obtuse angle, said lines lies in a plane which perpendicular to said first eccentric shaft [5], in the other word, if dividing the plane into I, II, III and IV four quadrants by the axle center of said driving gear shaft [17], then the axle center of said first eccentric shaft [5] and the axle center of said second eccentric shaft [8] will not lie in a same quadrant.

A front end portion of said pushing member [9] is against upon a lower rear portion of said

lifting member [11], a upper front end portion of said lifting member [11] is against upon said sleeve bearing [12]. A spring [2] providing between said housing [13] and said sleeve bearing [12]. Said first eccentric shaft [5] and said second eccentric shaft [8] respectively provided on the top surface and the bottom surface of said driving gear [7].